B.E. Seventh Semester (Mechanical Engineering / Power Engineering) (C.B.S.)

## **Energy Conversion - II**

# P. Pages : 3

1.

Time : Three Hours

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NKT/KS/17/7475/7521

Max. Marks: 80

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- Notes: 1. All questions carry marks as indicated.
  - 2. Solve Question 1 OR Questions No. 2.
  - 3. Solve Question 3 OR Questions No. 4.
  - 4. Solve Question 5 OR Questions No. 6.
  - 5. Solve Question 7 OR Questions No. 8.
  - 6. Solve Question 9 OR Questions No. 10.
  - 7. Solve Question 11 OR Questions No. 12.
  - 8. Due credit will be given to neatness and adequate dimensions.
  - 9. Assume suitable data whenever necessary.
  - 10. Illustrate your answers whenever necessary with the help of neat sketches.
  - 11. Use of non programmable calculator is permitted.
    - 12. Use of steam table, refrigeration table is permitted.
- a) What is the need of multistage compression & state its advantages.
  - b) A single stage double acting air compressor delivers air at 7.5 bar. The pressure & temperature at the end of suction stroke are 1 bar and 25°C. It delivers 2.2 m<sup>3</sup> of free per minute at 310 rpm. The clearance volume is 5% of stroke volume. The pressure and temperature of ambient air are 1.03 bar and 20°C. Take index of compression and expansion are 1.25 and 1.3 respectively. Determine : (1) Volumetric efficiency (2) Diameter and stroke length of cylinder if both are equal. (3) I.P. & B.P., if mechanical efficiency is 80%.

## OR

2. a) Prove that the volumetric efficiency of a single stage compressor is given by  $\eta_{vol} = 1 + C - C(r_p)^{1/n}$ 

Where, C = Clearance ratio,  $r_p = Pressure Ratio \& n = polytropic index$ .

- b) A two stage reciprocating air compressor delivers air at 17.25 bar has clearance volume of L. P. cylinder = 4% of swept volume. The atmospheric pressure is 1 bar and temperature is 15°C. At the start of compression the pressure in cylinder is 0.98 bar & 32°C. Intercooling pressure in cylinder is 4 bar. Take index of compression & expansion as 1.25 Find :
  - a) Work done per kg.
  - b) Volumetric efficiency refer to atmospheric condition. Assume P atm = 1. bar T atm =  $25^{\circ}$ C.
  - c) Isothermal efficiency
  - d) Saving in work as compared with single stage.
- **3.** a) Explain with the help of neat sketch and p v diagram a Root air blower.
  - b) A centrifugal compressor used as a super charger for aero engines handles 150 kg/min of air. The suction pressure & temperature are 1 bar and 290 K. The suction velocity is 80 m/s. After compression in the impeller the conditions are 1.5 bar, 345 K and 220 m/s. Calculate :
    - i) Isentropic efficiency
- ii) Power required to drive the compressor
- iii) The overall efficiency of the unit

It may be assumed that K.E. of air gained in the impeller is entirely converted into pressure in the diffuser.

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OR

P.T.O

Differentiate between axial flow compressor and centrifugal compressor.

In an eight stage axial flow compressor, the overall stagnation pressure ratio achieved is 5:1 with an overall isentropic efficiency of 92%. The mean blade speed is 160 m/s and 50% reaction design is used. The axial velocity through the compressor is constant and is equal to 90 m/s. The inlet stagnation temperature & pressure at inlet are 290 K & 1 bar. The work is divided equally between the stages.

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Calculate :

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a)

b)

- i) The blade angles
- ii) The power required.
- a) Differentiate between two stroke & four stroke engines.
  - b) Explain the stages of combustion in SI engines with the help of P  $\theta$  diagram.
  - c) Explain with neat sketch the working of simple carburetor. States its limitations.

#### OR

- a) Differentiate between SI & CI engines.
  - b) Explain Morse test method to determine I.P. of multi cylinder engine.
  - c) Explain the phenomenon of knocking in S.I. Engines.
- 7. a) Describe the laboratory method used for measuring the air supplied to an I.C. engines.
  - b) The power output of a six cylinder four stroke engine is measured by a hydraulic dynamometer for which the law is  $\frac{WN}{20,000}$ , where W is in Newton and the speed N is in rpm.

The air consumption is measured by air box method. The following reading are obtained : Orifice diameter = 30 mm, Stroke = 120 mm

bore = 100 mm, brake load = 560 N.

Coefficient of discharge of orifice = 0.6

ambient pressure = 1 bar, pressure drop across orifice = 14.5 cm of hg.

Time taken for  $100 \text{ cm}^3$  of fuel consumption is 20 sec, ambient temperature =  $27^{\circ}$ C,

fuel density =  $835 \text{ kg}/\text{m}^3$ , Speed = 2400 rpm.

- Calculate :
- (a) B.P.
- (b) B.S.F.C.
- (c) Volumetric efficiency.

#### OR

8.

During test of four stroke, eight cylinder petrol engine the following data was obtained : Load on hydraulic dynamometer = 950 N, Speed = 1250 rpm, bore = 70 mm, stroke = 89 mm, fuel consumption = 19.5 kg/hr, Calorific value of fuel used = 43 MJ/kg. Jacket water flow = 1400 kg/hr, Rise in water jacket temperature = 30°C, Air fuel ratio = 32, Room temperature = 30°C. exhaust gas temperature = 400°C, fuel analysis : 15% H<sub>2</sub> & 85% C, Specific heat of vapour =2.09 KJ/kgK, Partial pressure of water vapour in exhaust gas= 0.02 bar, specific heat of gases = 1.005 KJ/kgK, Constant of hydraulic dynamometer = 17280. Draw heat balance sheet in KJ/sec. & % basis.

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Explain the effect of superheating & subcooling on a simple vapour compression refrigeration cycle.

b) In a vapour compression refrigeration system the data available is as follows : Diameter of 2 cylinder single acting compressor = 15 cm. Length of stroke = 14 cm,  $\eta_r = 80\%$ , R.P.M. = 250, evaporator & condenser temperature are  $-15^\circ$ C & 30°C. Liquid ammonia enters expansion valve at 21°C. Find : (i) Refrigeration capacity of plant (ii) C.O.P.

Take :  $Cp_{liquid} = 4.6 \text{ KJ} / \text{kgK}$ 

 $Cp_{vapour} = 2.46 \text{ KJ} / \text{kgK}$ ,

Use following table.

a)

Temperat	Specific	$h_{f}$	hg	S <sub>f</sub>	Sg
ure (°C)	volume m <sup>3</sup> /kg	(KJ/Kg)	(KJ/Kg)	(KJ/KgK)	(KJ/KgK)
- 15	0.509	112.3	1426	0.457	5.549
30	0.111	323.1	1469	1.204	4.984

#### OR

- 10. a) Explain with neat sketch the simple vapour absorption refrigeration system.
  - b) A R-12 installation has following data.

Temperature of refrigerant superheated as gas in the evaporator =  $-5^{\circ}$ C Capacity : 15TR, evaporator temperature =  $-10^{\circ}$ C

Condenser temperature : 30°C

Temperature of refrigerant subcooled as liquid in the condenser : 25°C

Compressor particulars : No. of cylinder = 2

bore is 1.5 times stroke, rpm is 960

Determine :

- a) Mass of refrigerant circulated per minute.
- b) COP
- c) Volumetric efficiency, piston displacement, bore & stroke of the compressor; if the clearance volume is 3% of stroke volume.
- **11.** a) Define the following terms related with moist air.
  - i) Degree of saturation
- ii) Wet bulb temperature
- iii) Relative humidity
- iv) Specific humidity
- v) Dew point temperature
- b) With the help of steam table; determine (i) partial pressure of water vapour, (ii) dew point temperature, (iii) relative humidity, (iv) specific humidity & (v) specific enthalpy; if dry bulb temperature reads 40°C and wet bulb temperature is 30°C. Barometer reads 740 mm of Hg.

### OR

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12. a) In a cooling application, moist air enters refrigeration coil at a rate of 100 kg of dry air per minute at 35°C & 50% RH. The apparatus due point of coil is 5°C & by pass factor is 0.15. Determine the outlet state of moist air and cooling capacity of coil in T.R.

b) Describe the detail of working of air washer.

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